

# Freezing droplet ejection by spring-like elastic pillars

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Preventing water droplet accretion on surfaces is fundamentally interesting and practically important. Water droplets at room temperature can spontaneously detach from surfaces through texture design or coalescence-induced surface-to-kinetic energy transformation. However, under freezing conditions, these strategies become ineffective owing to the stronger droplet–surface interaction and the lack of an energy transformation pathway. Leveraging water volume expansion during freezing, we report a structured elastic surface with spring-like pillars and wetting contrast that renders the spontaneous ejection of freezing water droplets, regardless of their impacting locations. The spring-like pillars can store the work done by the seconds-long volume expansion of freezing droplets as elastic energy and then rapidly release it as kinetic energy within milliseconds. The three-orders-of-magnitude reduction in timescales leads to sufficient kinetic energy to drive freezing droplet ejection. We develop a theoretical model to elucidate the factors determining the successful onset of this phenomenon. Our design is potentially scalable in manufacturing through a numbering-up strategy, opening up applications in deicing, soft robotics and power generation.

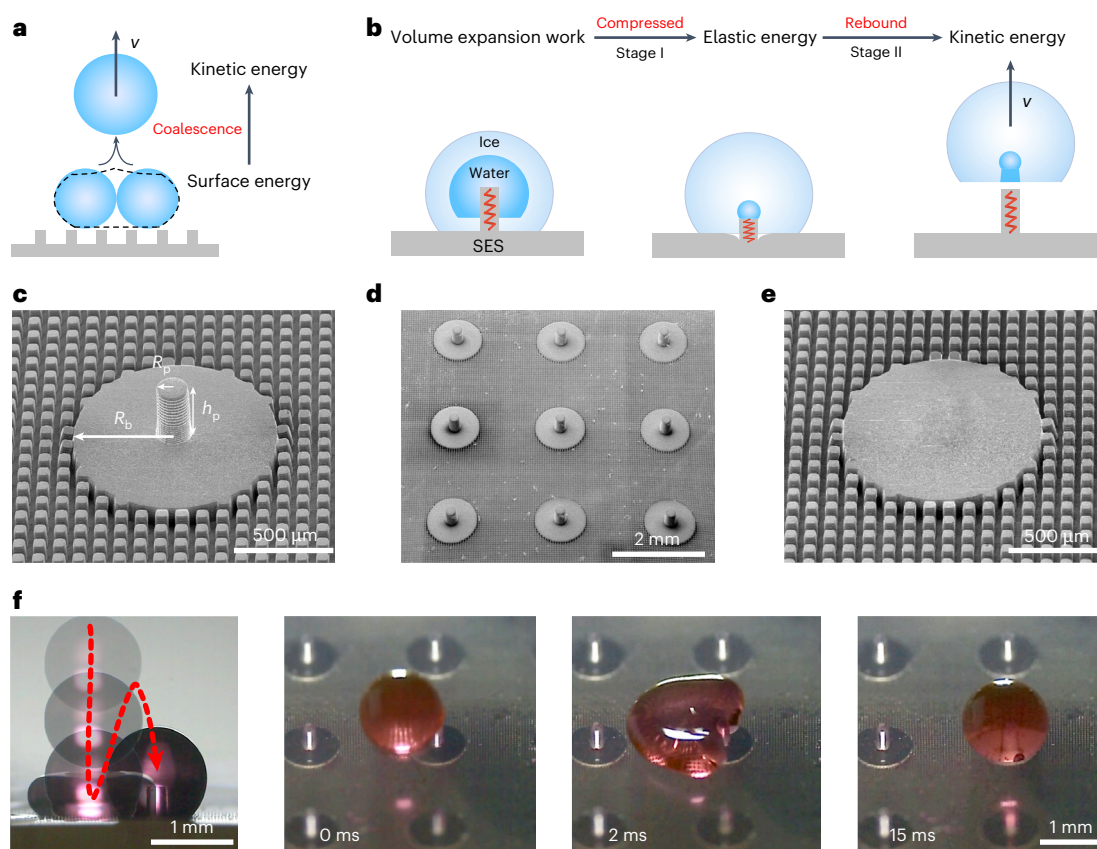
Timely and spontaneous removal of droplets from surfaces is desired in many practical applications such as energy harvesting<sup>1</sup>, thermal management<sup>2</sup>, self-cleaning<sup>3</sup> and anti-icing<sup>4,5</sup>. Fueled by evolutionary innovations, nature has mastered this skill, as exemplified by the repelling of water droplets on lotus leaves<sup>6</sup>, pitcher plants<sup>7,8</sup> and cicada wings<sup>9</sup>, as well as the shedding of sticky honeydew droplets on galling aphids<sup>10</sup>. Over the past decade, extensive efforts have been made to develop nature-inspired surfaces<sup>3,8,11</sup>, on which water droplets remaining at above-freezing conditions can be rapidly removed via the utilization of surface energy, such as coalescence-induced jumping<sup>9,12</sup> and bouncing<sup>11,13–16</sup> (Fig. 1a). But such surface-energy-driven strategies become ineffective for freezing water droplets in sub-freezing environments. This is because, in striking contrast to water droplets at room temperature, the freezing droplets have a much stronger adhesion

to underlying surfaces, and the formation of an ice shell prevents the coalescence of droplets for surface energy release.

Modifying surfaces with proper wettability<sup>17–19</sup>, stiffness<sup>20,21</sup> or slipperiness<sup>4,22–24</sup> can delay ice nucleation and propagation or reduce ice adhesion<sup>25–28</sup>; however, external energy is still needed to remove freezing droplets before they accrete into large ice films. For example, under vacuum conditions, droplets freezing on superhydrophobic surfaces can exhibit self-expulsion behavior by leveraging the explosive and asymmetric release of latent heat to generate a recalescence force to overcome the surface adhesion<sup>29</sup>. However, at atmospheric pressure, where the recalescence force becomes negligible compared with the adhesion force, the freezing droplets remain adhered to the underlying surfaces instead of being expelled<sup>29</sup>. The accretion of freezing droplets and icing can lead to detrimental impact, as evidenced by the disaster

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**Fig. 1 | Design of the structured elastic surface.** **a**, Schematic showing water droplet ejection driven by the surface energy released from the coalescence of two contacting droplets. **b**, Conception of freezing droplet ejection on an SES that can harness the volume expansion work of the freezing droplet through a two-stage energy conversion process. **c**, Scanning electron microscopy image showing the structures of the SES, consisting of a pillar with radius  $R_p$  and height

$h_p$ , a smooth base with radius  $R_b$  and a micropatterned substrate surrounding the base. **d**, Scanning electron microscopy image of the SES arrays. **e**, Scanning electron microscopy image showing the non-structured surface as a control sample. **f**, Optical images showing the automatic positioning of a randomly released droplet onto a pillar on SES arrays.

in Texas in 2021 that disrupted high-speed roads and power lines, troubling millions of families and businesses<sup>30</sup>. So far, achieving freezing droplet self-removal from surfaces remains challenging despite encouraging recent advances in understanding freezing dynamics from the ice nucleus to the film level<sup>30–33</sup>.

One simple fact is that the freezing of water naturally leads to ~9% volume expansion under normal atmospheric conditions<sup>34</sup>, which might serve as an alternative energy source for droplet ejection. Previously, it has been shown that the volume expansion work done by a small volume of freezing water can produce high local pressure that even fractures brittle cast iron, known as a freezing bomb<sup>34</sup>. However, owing to the slow freezing process, utilizing the volume expansion work to dynamically render the spontaneous ballistic ejection of droplets is more challenging than statically fracturing cast iron. Although previous efforts have successfully utilized the volume expansion effect to achieve the self-dislodging of freezing droplets on superhydrophobic surfaces, the self-dislodging freezing droplets exhibit limited motility under atmospheric conditions<sup>19</sup>, highlighting the challenges in utilizing volume expansion work to drive droplet ejection.

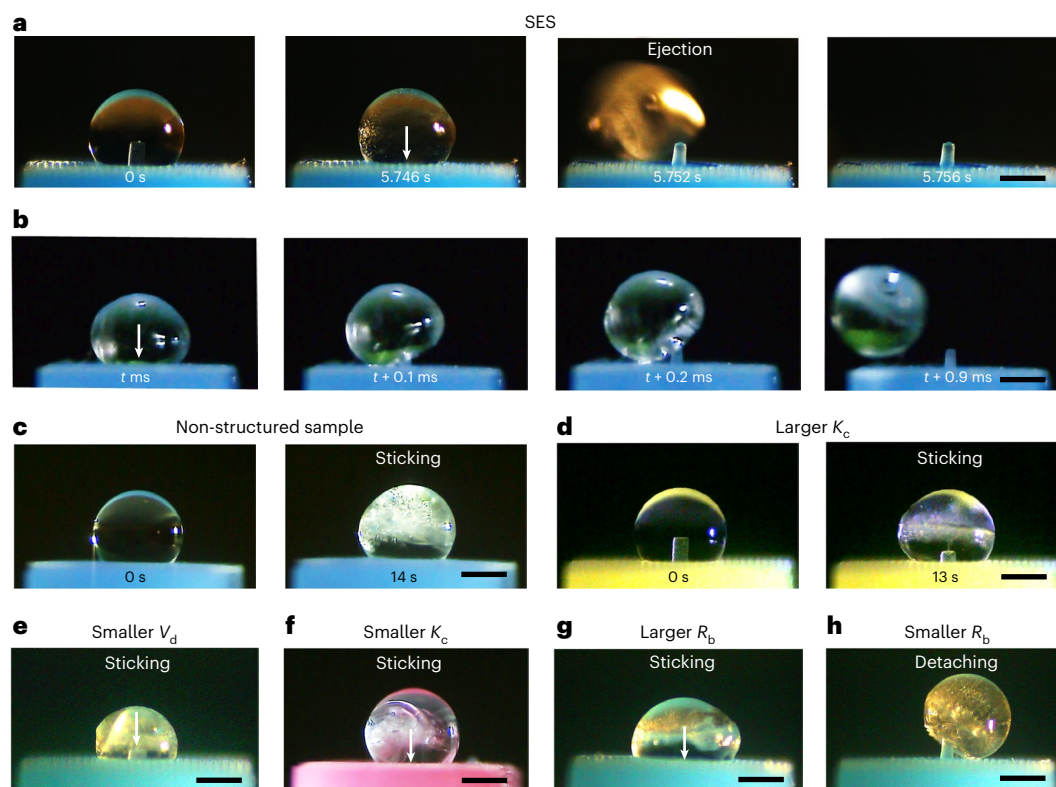
To accelerate energy transformation to drive freezing droplet ejection, we conceive that by the rational design of a structured elastic surface (SES), it is possible to first convert the freezing droplet's volume expansion work into the elastic energy stored in the SES (stage I) and then into the kinetic energy of the freezing droplet in a rapid manner (stage II) (Fig. 1b and Supplementary Video 1). One intriguing analogous example achieving such an elastic-to-kinetic energy transformation is the fungus *Pilobolus kleinii*, which can spontaneously disperse spores

(Extended Data Fig. 1)<sup>35,36</sup>. Briefly, its vesicle stores elastic energy via the absorption of ambient moisture. Once a critical internal pressure is reached, the vesicle explodes and rapidly releases the stored elastic energy, ballistically discharging the spores at its tip<sup>35</sup>.

## Results

### Device design and fabrication

Figure 1c shows a representative SES that consists of three key components: a smooth base with radius  $R_b = 0.5$  mm, a deformable spring-like pillar with radius  $R_p = 0.09$  mm and height  $h_p = 0.4$  mm located at the center of the base, and a micropatterned substrate surrounding the base (Fig. 1c). The SES is made of elastic polydimethylsiloxane (PDMS) through a simple cast-molding process (Extended Data Fig. 2a), allowing large-scale production as evidenced by the SES arrays (Fig. 1d). Moreover, we can easily tailor the compressive modulus  $K_c$  of the SES through the mixing ratio of base to crosslinker of PDMS (Extended Data Fig. 2b and Extended Data Table 1). We also prepared two control samples, namely, a non-structured sample without the pillar structure (Fig. 1e) and a stiffer sample with larger  $K_c$ . Note that owing to the intrinsic hydrophobicity of PDMS, the contact angle of a water droplet on the micropatterned substrate is amplified to ~160°, indicating the superhydrophobicity of the micropatterned substrate (Extended Data Fig. 3). The wettability contrast between the hydrophobic smooth base and the superhydrophobic micropatterned substrate of SES can drive randomly impacting droplets to spontaneously move to the smooth base and be captured by the spring-like pillar, which facilitates potential practical use of the designed surface.



**Fig. 2 | Comparison of the freezing droplet behaviors on SES and control samples.** **a**, Spontaneous ejection of a 1  $\mu\text{l}$  freezing droplet on an SES with  $R_b = 0.5$  mm and  $K_c = 1.7$  MPa. **b**, High-speed imaging of the ejection process of a 1  $\mu\text{l}$  freezing droplet on the SES sample used in **a**. **c, d**, Sticking of 1  $\mu\text{l}$  freezing droplets on the non-structured sample without the pillar (**c**) and the stiffer sample with  $K_c = 5.9$  MPa (**d**). **e–h**, Dependence of freezing droplet behaviors on droplet size, SES stiffness and SES geometry. **e**, Sticking of a smaller freezing droplet (0.4  $\mu\text{l}$ ) on the SES sample used in **a**. **f**, Sticking of a 1  $\mu\text{l}$  freezing droplet

on the SES with a smaller compressive modulus  $K_c$  (0.6 MPa) than the one used in **a**. **g**, Sticking of a 1  $\mu\text{l}$  freezing droplet on the SES sample with a larger  $R_b$  (0.7 mm) than the one used in **a**. **h**, Detaching of a 1  $\mu\text{l}$  freezing droplet on the SES sample with a smaller  $R_b$  (0.2 mm) than the one used in **a**. Scale bars, 500  $\mu\text{m}$ . The color of the SES represents the compressive modulus, with yellow corresponding to 5.9 MPa (**d**), blue to 1.7 MPa (**a–c**, **e**, **g** and **h**), and pink to 0.6 MPa (**f**). The arrows indicate the positions of deformed pillars.

As shown in Fig. 1f, a water droplet impacting the SES arrays spreads and contacts with a nearby pillar, retracts asymmetrically toward the pillar and finally settles on top of it.

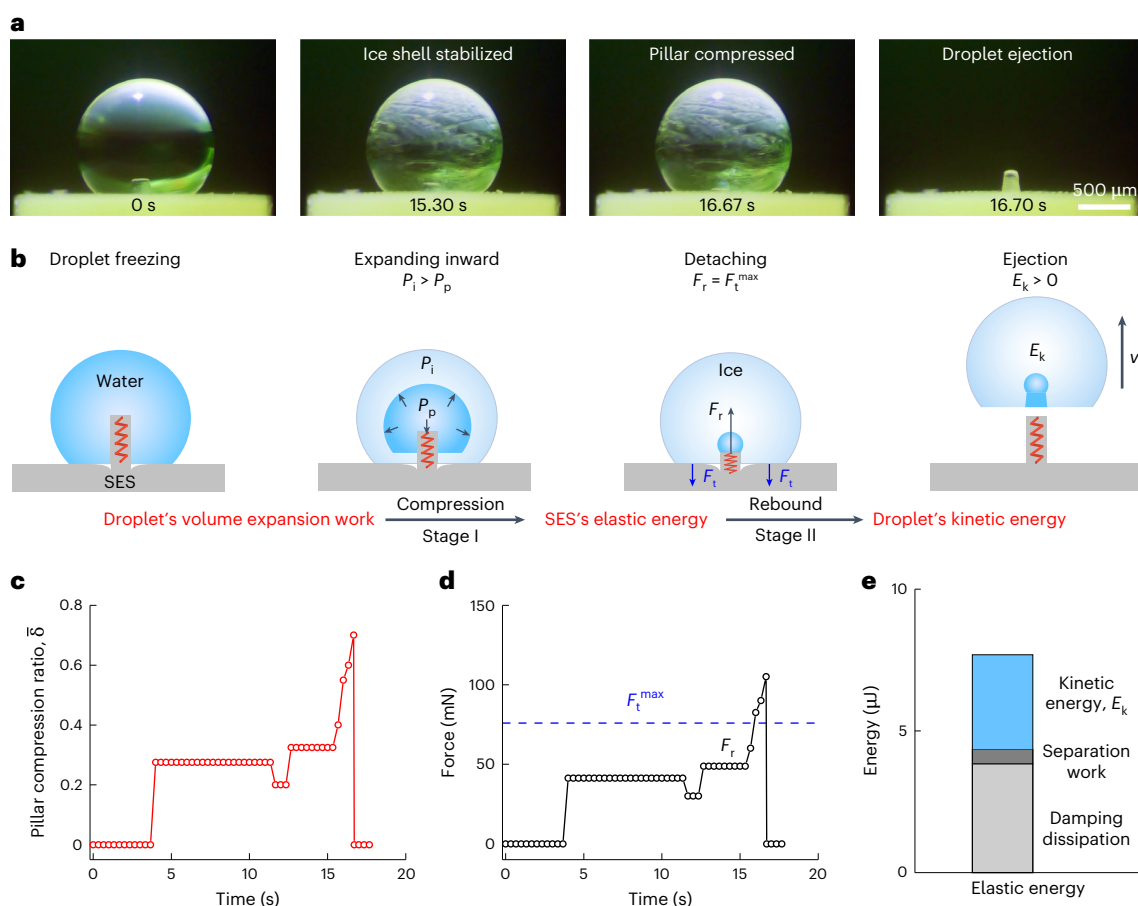
### Droplet ejection observations

Figure 2 compares the behaviors of droplets freezing on the SES and the two control samples. To mimic natural freezing conditions, we conducted the experiments in a customized isothermal chamber (Extended Data Fig. 4), where the droplet is cooled down by the circulated cold air at  $-15$   $^{\circ}\text{C}$  rather than the underlying substrate, as shown by thermographic imaging (Extended Data Fig. 5 and Supplementary Video 2). The isothermal condition allows droplets to freeze in an inward manner. On the SES sample with  $R_b = 0.5$  mm and  $K_c = 1.7$  MPa, a 1  $\mu\text{l}$  water droplet gradually freezes in the first 5 s and is suddenly ejected away at 5.752 s (Fig. 2a). By recording the ejection process at 10,000 frames per second (fps), we found that the compressed spring-like pillar rapidly bounces up within 0.2 ms, which probably gives rise to sufficient force and energy for freezing droplet ejection (Fig. 2b). The ejection trajectory randomly deviates from the vertical direction, which is mainly attributed to the non-perfectly symmetrical geometry and freezing process of the droplet in practice. Such a non-vertical trajectory enables the droplet to escape from the surface rather than return to the original launching site. By contrast, for the non-structured or stiffer samples, the 1  $\mu\text{l}$  droplets completely freeze and stick to these surfaces (Fig. 2c,d). Such contrasting phenomena suggest that the pillar structure and the stiffness of SES are essential for achieving the spontaneous ejection of freezing droplets.

We further experimentally evaluate the influences of droplet size, SES stiffness and SES base size on the spontaneous ejection behaviors of freezing droplets. On the same SES sample used in Fig. 2a, we observed that a smaller droplet (0.4  $\mu\text{l}$ ) did not exhibit the spontaneous ejection behavior, but stuck to the SES after freezing (Fig. 2e). Alternatively, on an SES sample with the same structural dimensions but smaller  $K_c$  (0.6 MPa), a 1  $\mu\text{l}$  freezing droplet displayed no spontaneous ejection behavior, but stuck to the base surface (Fig. 2f). Moreover, it was found that the advent of the spontaneous ejection behavior also depends on the base radius ( $R_b$ ) of the SES because SES samples with the same  $K_c$  and pillar as in Fig. 2a failed to eject a 1  $\mu\text{l}$  freezing droplet when the base radius was greatly enlarged or reduced, resulting in droplet sticking or detaching as the only consequences, respectively (Fig. 2g,h). These diverse phenomena indicate that the spontaneous ejection of freezing droplets necessitates a proper design of the structural and mechanical properties of the SES in response to droplets of different volumes.

### Droplet ejection mechanisms

To reveal the underlying mechanisms, we further visualize the dynamic interactions between the freezing droplet and the SES. To clearly distinguish water from ice, we added fluorescent dye to the water droplet. When water is frozen into ice with a crystal structure, fluorescent dye molecules will be expelled out of ice crystals and aggregate together, causing quenching behavior. Because of the aggregation-induced quenching, the ice portion becomes colorless, whereas the water part remains transparent green (Fig. 3a). The transparent water also



**Fig. 3 | Characterization and theoretical modeling of the dynamic interactions between the freezing droplet and the SES.** **a**, Visualization of a 2  $\mu\text{l}$  dyed freezing droplet ejection process on the SES sample with  $R_0 = 0.5$  mm and  $K_c = 5.9$  MPa. **b**, Schematic of the theoretical model illustrating the two-stage energy conversion process for freezing droplet ejection by considering the pillar as a spring for energy storage and release. The two-stage energy conversion process is shown: stage I, converting the droplet's volume expansion work into SES's elastic energy via pillar compression; stage II, converting SES's elastic energy into the droplet's kinetic energy via pillar rebound.  $P_i$ , the pressure-bearing capacity of the ice shell of the freezing droplet;  $P_p$ , the internal pressure of the freezing

droplet;  $F_r$ , the repelling force generated by the compressed pillar;  $F_t^{\text{max}}$ , the maximal traction force;  $E_k$ , the kinetic energy gained by the freezing droplet. **c**, Real-time pillar compression ratio  $\bar{\delta}$  normalized to the initial height of the pillar. **d**, Real-time analysis of the upward repelling force  $F_r$  exerted by the compressed pillar and the maximal traction force  $F_t^{\text{max}}$  on the freezing droplet. **e**, Quantitative analysis of the energy flow for the ejection behavior. At the energy release stage, part of the stored elastic energy  $E_e$  is dissipated by the damping and separation work, and the remaining part is converted into the kinetic energy  $E_k$  of the freezing droplet.

allows us to examine the dynamics of the pillar inside the droplet. We experimentally confirmed that the addition of fluorescent dyes did not affect the behavior of the frozen droplets, as no behavioral differences between the dyed and undyed droplets were observed. At the initial freezing stage, the ice shell of the droplet is unstable, easily breaking at the weakest point and then re-freezing on cooling by the chilled air, repeatedly until a stable shell is formed. Once the seamless ice shell encloses the freezing droplet, its further volume expansion starts to compress the elastic pillar downward via the incompressible water medium (Fig. 3a). When the pillar is compressed by about 70% of its original height, the freezing droplet is suddenly ejected away from the surface within milliseconds at a velocity of  $-1$  m  $\text{s}^{-1}$  (Fig. 3a).

The dynamics of droplet ejection on SES are also sensitive to freezing conditions, such as the environmental temperature that influences the freezing rate and therefore the compression rate of the spring-like pillar. To showcase the effect of environmental temperature on droplet ejection, we performed droplet-freezing experiments at two distinct temperatures of  $-15$   $^{\circ}\text{C}$  and  $-5$   $^{\circ}\text{C}$ . It is shown that the droplet ejection phenomenon still occurs at an elevated temperature of  $-5$   $^{\circ}\text{C}$ , but at a much slower rate (Supplementary Video 3). This is mainly because at a relatively higher temperature, the freezing rate is reduced, which

delays the formation of a stable ice shell and slows down the compression rate of the spring-like pillar. For quantitative analysis, we measured the real-time compression ratio of the pillar and calculated the average compression rate (Extended Data Fig. 6), which is also equal to the volume expansion rate of the droplet as the volume of the enclosed frozen droplet is conserved. The average compression rate for  $-5$   $^{\circ}\text{C}$  is only  $0.42$   $\text{s}^{-1}$ , one-fifth that for  $-15$   $^{\circ}\text{C}$  ( $2.14$   $\text{s}^{-1}$ ), suggesting the influences of freezing heat transfer on the spring compression process and, thus, the minimum time required for ejection occurrence.

The ejection behavior of the freezing droplet on SES results from a two-stage energy conversion process consisting of energy storage (stage I) and energy release (stage II). Serving as an energy harvester, the SES first stores the volume expansion work the freezing droplet does as elastic energy over seconds and then rapidly releases the stored elastic energy into the droplet's kinetic energy (Fig. 3a,b). The duration of the energy release ( $-1$  ms) is approximately three orders of magnitude shorter than that of the energy storage ( $-1,000$  ms), indicating an amplification of mechanical power output for propelling freezing droplet ejection. The energy storage begins with the downward compression of the spring-like pillar of the SES, caused by the water-to-ice volume expansion in the freezing droplet enclosed by the

ice shell (Fig. 3b). The compressed pillar generates a repelling force that grows with further compression. When the repelling force  $F_r$  is able to overcome the maximum traction force  $F_t^{\max}$ , the freezing droplet detaches from the surface, and at the same time, the elastic energy stored in the pillar is rapidly released and partially converted into the droplet's kinetic energy. The amount of kinetic energy  $E_k$  ultimately gained by the freezing droplet determines whether the ejection behavior occurs. To probe how the droplet size and the stiffness and base size of the SES govern the two-stage energy conversion process, we built a theoretical model by treating the SES as a spring-like mechanical gadget for energy storage and release.

Considering the importance of pillar compression in energy storage and release, we first evaluate the theoretical maximal compression ratio during the freezing process of a given droplet. In stage I of the two-stage energy conversion process, the pillar may undergo multiple cycles of compression and rebound until the droplet detaches from or fully freezes on the SES (Fig. 3a). The time between two adjacent compression cycles depends on the healing speed of the fractured ice shell, which is related to the cooling conditions, such as the temperature and velocity of the chilled air flow. In a single cycle, the pillar is continuously compressed from its original height until the ice shell of the freezing droplet is fractured by the increased internal pressure owing to the continuous inward volume expansion, or until the freezing droplet detaches (Fig. 3a,c).

Fundamentally, pillar compression is governed by the competition between the pressure-bearing capacity  $P_i$  of the ice shell and the internal pressure  $P_p$ . We calculate that  $P_i = \sigma_i[(1 - \phi)^{-\frac{2}{3}} - 1]$  (see the calculation of the maximal compression ratio of the pillar in the Methods), where  $\sigma_i$  is the fracture strength of ice ( $\sim 0.3$  MPa)<sup>37</sup>, and the volume fraction of the solidified water  $\phi$  reflects the progression of the freezing process with  $\phi = 0$  referring to the non-frozen state and  $\phi = 1$  indicating the fully frozen state of the droplet. The internal pressure  $P_p$  can be correlated with the pillar's compression ratio through  $P_p = K_c \delta$ , where  $\delta$  is the compression ratio defined as the downward displacement  $\delta$  of the pillar tip relative to the original height of the pillar  $h_p$ . Because the volume conceded by the descending pillar is equal to the newly expanded volume of the freezing droplet within one cycle, we have  $\delta = 0.09V_d(\phi_1 - \phi_0)/V_p$ , where  $\phi_0$  and  $\phi_1$  denote the extent of volume expansion at the starting and ending points of the cycle, respectively (Extended Data Fig. 7; see the calculation of the maximal compression ratio of the pillar in the Methods). Note that this volume relationship between droplet expansion and pillar compression only applies to the outside-in freezing directionality where the freezing front acts as a capsule to encapsulate the unfrozen water, and does not apply to other freezing directionalities (top-down, bottom-up or sideways). In the last cycle, during which  $P_i \geq P_p$  is always guaranteed, the maximal compression ratio  $\delta_{\max}$  is attained. Together with the constraint that  $\delta_{\max}$  should be no larger than unity (otherwise draining occurs and the whole process halts), we have

$$\delta_{\max} = \min \left\{ \frac{0.09V_d}{V_p} (1.96\mu^{-3/5} - \mu^{-1}), 1 \right\}, \quad (1)$$

where  $\mu = 0.09 \frac{V_d}{V_p} \frac{K_c}{\sigma_i}$  (see the calculation of the maximal compression ratio of the pillar in the last cycle in the Methods). As shown in Extended Data Fig. 8, as  $V_d/V_p$  increases, the maximal compression ratio  $\delta_{\max}$  first increases and then reaches a plateau of 1. Additionally,  $\delta_{\max}$  exhibits a negative dependence on  $K_c$  under a given  $V_d/V_p$  (Extended Data Fig. 8).

Next, we resort to force analysis to determine whether the stored elastic energy in the pillar can be released. The freezing droplet is subject to two directionally opposite forces, including the downward traction force  $F_t$  composed of intermolecular adhesion and pressure difference and the upward repelling force  $F_r$  exerted by the spring-like pillar via incompressible water as the medium. Here the gravitational force is found to be 2–3 orders of magnitude lower than the maximum

traction force and the repelling force and is, thus, not included. The maximum traction force  $F_t$  is estimated to be  $F_t^{\max} \approx \pi P_{\text{atm}}(R_b^2 - R_p^2)$  (Extended Data Fig. 9; see the calculation of the maximal traction force in the Methods), where  $P_{\text{atm}}$  denotes the atmospheric pressure and is a constant value of 0.1 MPa. The absence of intermolecular adhesion force in  $F_t^{\max}$  can be attributed to the fact that at the critical detaching moment, a large fraction of the originally contacting region has deboned and is dominated by the vacuum region. The upward repelling force generated by the spring-like pillar is proportional to the compression ratio through  $F_r = \pi R_p^2 K_c \delta$ , with a maximal value at  $\delta = \delta_{\max} \leq 1$ . According to the force equilibrium principle, if the maximal  $F_r$  can overcome  $F_t^{\max}$ , the freezing droplet can detach from the base surface to release the stored elastic energy. Otherwise, it will stick to the base surface. As shown in Fig. 3d, when the pillar reaches a compression ratio at which  $F_r$  exceeds  $F_t^{\max}$ , the droplet detachment is successfully triggered. On the basis of the condition that the maximal  $F_r$  should be no smaller than  $F_t^{\max}$ , we can derive the key parameters that satisfy the force criterion for droplet detachment, expressed as

$$\frac{K_c}{P_{\text{atm}}} \geq \frac{R_b^2}{R_p^2} - 1. \quad (2)$$

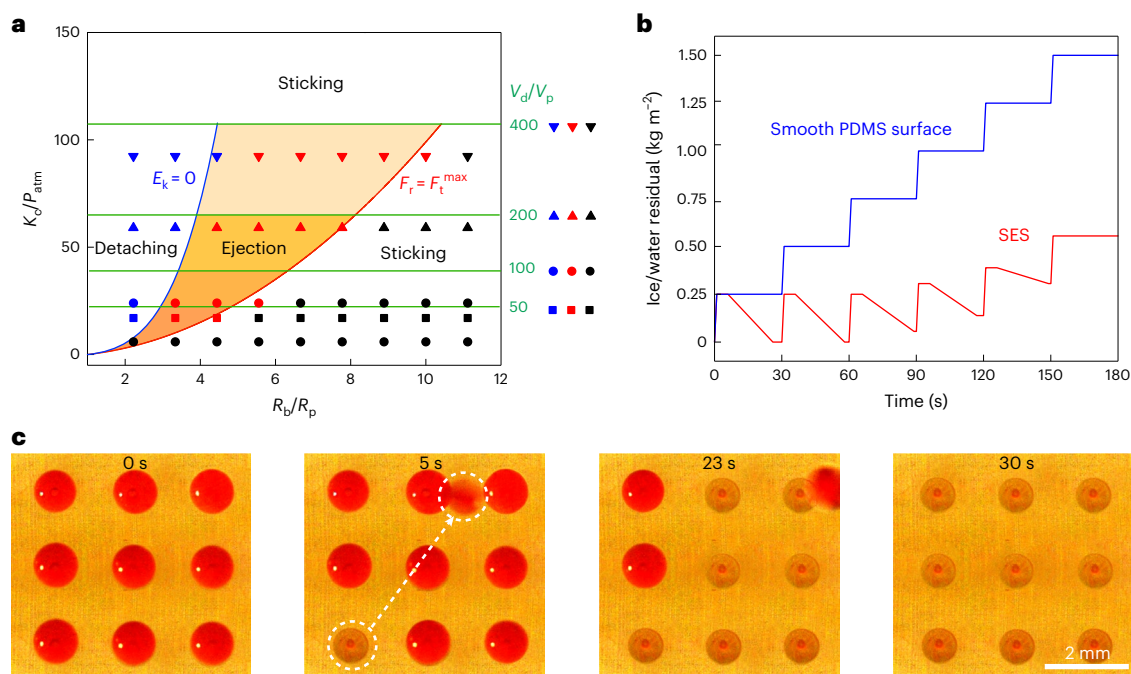
Thus, we can obtain the lower bound of  $K_c/P_{\text{atm}}$  as a function of  $R_b/R_p$ , above which the repelling force can overcome the traction force, triggering droplet detachment, and thus, the elastic energy can be released.

Achieving the ejection of freezing droplets also requires attaining sufficient kinetic energy during the release of the stored elastic energy. In stage II of the two-stage energy conversion process, the released elastic energy  $E_e$  has first to pay off the damping dissipation of the pillar  $E_d$  and the work for separating the droplet from the pillar  $E_s$ . At the critical detaching moment of the droplet, the amount of elastic energy stored in the spring-like pillar is expressed by  $E_e = \frac{K_c}{2} V_p \delta_{\max}^2 = \frac{K_c}{2} V_p \left[ \frac{P_{\text{atm}}}{K_c} \left( \frac{R_b^2}{R_p^2} - 1 \right) \right]^2$  (see the energy analysis in the Methods). The damping dissipation of the pillar elastic energy and the separation work are estimated to be  $E_d = (1 - \xi)E_e$  and  $E_s = P_{\text{atm}} V_p \left[ 1 - \frac{P_{\text{atm}}}{K_c} \left( \frac{R_b^2}{R_p^2} - 1 \right) \right]$  (see the energy analysis in the Methods), respectively, where  $\xi$  is the rebound resilience of the pillar, defined as the ratio of energy released from the deformation recovery to the stored strain energy and measured as  $\sim 50\%$  for the PDMS materials (Extended Data Table 1). As a result, the ultimate kinetic energy gained by the freezing droplet is  $E_k = \xi E_e - E_s$ . For the case shown in Fig. 3a,  $E_k$  is calculated as 3.3  $\mu\text{J}$  and the corresponding energy conversion efficiency is estimated to be 0.435 (Fig. 3e), which explains the occurrence of freezing droplet ejection. In a more general picture, we can write the energy criterion as

$$\frac{K_c}{P_{\text{atm}}} \leq \frac{1}{2} \xi \left( \frac{R_b^2}{R_p^2} - 1 \right)^2 + \left( \frac{R_b^2}{R_p^2} - 1 \right). \quad (3)$$

In addition to the force and energy conditions above, another necessary condition for ejection, which is easily neglected, is the consistent presence of liquid water in the ice shell until it separates from the pillar. Otherwise, the ice shell will be stuck to the pillar during the process for the depletion of liquid water even after it detaches from the base. To avoid water depletion, we consider the most conservative scenario, in which the volume expansion of the residual water at the beginning of the last cycle is enough to expel the pillar out from the ice shell, leading to (see the minimum droplet volume for preventing water depletion in the Methods)

$$\frac{K_c}{P_{\text{atm}}} \leq 1.224 \left( \frac{V_d}{V_p} - 11.11 \right)^{0.732} + 2. \quad (4)$$



**Fig. 4 | Phase diagram of the freezing droplet ejection on SES and practical application.** **a**, Phase diagram for the ejection of the freezing droplet on the SES. In this phase diagram, different volume ratios of the droplet to the pillar are taken:  $V_d/V_p = 400, 200, 100$  and  $50$ . **b**, Comparison of the deicing performance

between the SES arrays and the smooth PDMS surface under multicycle icing tests. **c**, Optical images showing one typical cycle of the self-deicing behavior of the SES arrays. The droplets were dyed with a fluorescent dye.

Combining the preceding force and energy analyses enables us to build a phase diagram for the dynamic behaviors of the freezing droplets on SES (Fig. 4a). In this phase diagram, different volume ratios of the droplet to the pillar are taken:  $V_d/V_p = 400, 200, 100$  and  $50$ . For each  $V_d/V_p$  value, the phase diagram shows three regions corresponding to three different droplet behaviors, namely, sticking, detaching and ejection, depending on the radius ratio  $R_b/R_p$  and the normalized compressive modulus  $K_c/P_{atm}$  of the SES (Fig. 4a). In our experiments, the ratio  $R_b/R_p$  was tailored by varying  $R_b$  and keeping  $R_p$  constant ( $0.09$  mm). For SES with a constant  $K_c/P_{atm}$ , an excessively large  $R_b/R_p$  makes the droplet fail to overcome the traction force from the surface, resulting in the “sticking” behavior of the droplet. Although an excessively small  $R_b/R_p$  enables the droplet to overcome the surface traction, the stored elastic energy in the compressed pillar is insufficient for ejection, resulting in the “detaching” behavior only of the droplet. Only SES with a proper  $R_b/R_p$  satisfies both the force and the energy conditions, yielding the “ejection” behavior of the droplet, which accounts for the observations shown in Fig. 2g,h. Alternatively, for SES samples with a given  $R_b/R_p$  value, the behavior of the freezing droplet can be controlled by tuning  $K_c$  (Fig. 4a). This explains the failure of spontaneous ejection in the case shown in Fig. 2e. Comparing the phase diagrams for different  $V_d/V_p$  values indicates that the area of the ejection region depends on the droplet volume. The smaller the droplet, the smaller the ejection region. This is the probable explanation as to why a  $0.4$   $\mu\text{l}$  freezing droplet fails to be ejected by an SES that can eject a  $1$   $\mu\text{l}$  freezing droplet (Fig. 2d).

### Practical application potential

To demonstrate the practical application potential of this phenomenon, we conduct successive freezing experiments on the SES and find that the SES can eject two freezing droplets in sequence, demonstrating the chain ejection capability (Supplementary Video 4). To showcase the potential scalability of the SES in ejecting freezing droplets, we also conduct multiple cycles of icing tests on the SES with  $3 \times 3$  arrays as well as a smooth sample made of the same underlying materials.

In a single cycle, we deposit nine droplets on the two horizontally placed surfaces and measure the mass of residual ice after freezing for  $30$  s. After six consecutive cycles, we find that the ice residual on SES arrays grows slowly and is only  $\sim 33\%$  of that on the control sample (Fig. 4b). This is because the SES arrays have the ability to shed droplets away by converting the volume expansion work of freezing droplets into kinetic energy, reducing the ice residual on the surface (Fig. 4c and Supplementary Video 5).

### Discussion

In summary, we report droplet ejection by engineered structured elastic surfaces through a two-stage energy conversion process. Mediated by the spring role of the SES, the volume expansion work of freezing droplets is gradually stored as elastic energy at a typical timescale of seconds, whereas the stored elastic energy is released within milliseconds. Such a spring-mediated energy transformation gives rise to sufficient kinetic energy for droplet ballistic motion. Owing to the dependence of droplet ejection on the freezing front directionality, our design may not apply to the non-isothermal freezing ways, such as top-down freezing and bottom-up freezing (Supplementary Video 6), but is suited for common atmospheric-pressure isothermal freezing conditions, such as outdoor winter environments. The volume-expansion-work-induced droplet ejection deepens our understanding of the multiphase freezing dynamics for anti-icing applications. Our work also demonstrates a strategy that can efficiently harvest and exploit the volume expansion work of freezing droplets for ballistic motion generation, which could expand energy conversion phenomena<sup>38</sup> and inspire the development of droplet-based energy generators and soft robotic catapults<sup>39</sup>.

### Methods

#### Fabrication of SES sample

We fabricated the SES using a simple cast-molding process. First, a mold was made using a projection microstereolithography-based three-dimensional printing machine (BMF Nano Materials Technology), and coated with a thin gold layer via a Quorum Coater (Q150TS,

Quorum Technologies) to facilitate the demolding process. We then poured the PDMS (Dow Corning SYLGARD, PDMS 184) into the mold, degassed it for 30 min and cured it in an oven at 80 °C for 2 h. The SES sample was achieved by demolding and post-curing at 165 °C for 48 h. By changing the mold dimensions, we made SES samples with different pillar radii  $R_p$  and base radii  $R_b$ . Note that the pillar volume  $V_p$  (0.01  $\mu\text{l}$ ) was controlled to be the same for all the SES samples. We controlled the compressive modulus of the SES samples through the base/crosslinker ratios of PDMS. For comparison, we fabricated a non-structured sample using the same processes but with different molds.

### Mechanical characterization

We characterized the influences of the base/crosslinker ratio of PDMS on  $K_c$  and  $\xi$ . We select five base/crosslinker ratios by weight, including 3, 10, 15, 20 and 25. The  $K_c$  value of these SES samples was measured using an Instron 5567 universal testing machine. To measure the rebound resilience, cylindrical specimens made of PDMS with these five different ratios were tested using a Pendulum rebound tester (UTSTESTER, H029) based on the ASTM D 7121 standard.

### Visualization of freezing droplet ejection

To imitate natural cooling conditions in winter, we built a customized isothermal environmental chamber and visualized the freezing droplet behavior within. This chamber mainly consisted of a cooling tube and a cooling stage underneath the tube, connected using a heat-conductive glue (Hasuncast SG7630) for enhanced heat transfer between them. We placed the SES sample above the cooling tube, with a thermal insulation foam (Styrofoam) in between, to prevent direct heat transfer between the SES and the cooling tube. The cooling stage and tube temperatures were controlled at approximately -15 °C by the cooling circulator (Julabo, ED-F12) and the Peltier temperature control unit (Dataphysics Instruments, TPC 160). When the ambient air was pumped into the cooling tube, it was cooled down and circulated at 1  $\text{m s}^{-1}$  within the chamber. As a result, the droplet on the SES was cooled down by the circulated chilly air and started to freeze, like in natural winter environments. Under such cooling conditions, the droplet freezes inward from its periphery, forming an ice shell. The freezing process of the droplet and the associated thermal distribution dynamics were recorded using a high-speed camera (MEMRECAM HX-6, Nac) at a frame rate of 500 fps and a thermography camera (FLIR T1020) at a frame rate of 30 fps, respectively. We also filmed the droplet ejection process at a frame rate of 10,000 fps to provide more visual information.

### Calculation of the maximal compression ratio of the pillar in the last cycle

Pillar compression is dependent on the competition between the pressure-bearing capacity of the ice shell  $P_i$  and the internal pressure  $P_p$ . Assuming a compression-and-rebound cycle of the pillar starts at a freezing state with a fraction of the water solidified into ice  $\phi_0$ , the internal pressure and pressure-bearing capacity of the ice shell are given by  $P_p = 0.09K_c V_d (\phi - \phi_0) / V_p$  and  $P_i \approx \sigma_i [(1 - \phi)^{-\frac{2}{3}} - 1]$ , respectively, where  $\phi$  denotes the instant fraction of water solidified into ice ( $0 \leq \phi_0 \leq \phi \leq 1$ ). As shown in Extended Data Fig. 7, the progression of a compression-and-rebound cycle of the pillar depends on  $\phi_0$ . For a small  $\phi_0$ ,  $P_p$  would easily reach  $P_i$  as  $\phi$  grows, resulting in fracture of the ice shell and terminating the compression-and-rebound cycle of the pillar. After the healing of the fractured ice shell by the cooling environment, a new compression-and-rebound cycle starts from a larger  $\phi_0$ . Such a fracture-and-healing process will repeat until  $\phi_0$  is greater than a threshold. Subsequently,  $P_p$  would not exceed  $P_i$  for any  $\phi \in [\phi_0, 1]$ . Such a threshold of  $\phi_0$ , denoted by  $\phi_0^*$ , can be simply determined by finding the condition when curve  $P_p(\phi)$  becomes a tangent line of curve  $P_i(\phi)$  on the  $P/\sigma_i \approx \phi$  plane (Extended Data Fig. 7). We have

$$\phi_0^* = 1 - 1.96\mu^{-\frac{3}{5}} + \mu^{-1},$$

where  $\mu$  is a non-dimensional parameter  $\mu = 0.09 \frac{V_d K_c}{V_p \sigma_i}$ . In this cycle, the maximal compression ratio is achieved when  $\phi = 1$ , that is,

$$\delta_{\max} = 0.09V_d (1 - \phi_0^*) / V_p = \frac{0.09V_d}{V_p} (1.96\mu^{-3/5} - \mu^{-1}).$$

Meanwhile,  $\delta_{\max}$  should be no larger than 1 at which the pillar is fully compressed, leading to

$$\delta_{\max} = \min \left\{ \frac{0.09V_d}{V_p} (1.96\mu^{-3/5} - \mu^{-1}), 1 \right\}.$$

### Calculation of the maximal traction force

The downward compression of the pillar under internal pressure of  $P_p$  would gradually deform and separate the base of the SES from the freezing droplet, resulting in the formation and growth of a concave vacuum region with radius  $R_A$  (Extended Data Fig. 9a). At the same time, the annular contact area between the base and the freezing droplet, characterized by the ratio of  $R_A$  to  $R_b$ , would decrease. As a result, the downward traction force ( $F_t$ ) of the droplet is contributed by the contact adhesion (van der Waals force) and the suction induced by the pressure difference between the vacuum region and the atmosphere. On the basis of the Griffith criterion<sup>40</sup> for crack propagation and the solutions to stress intensity factors in an annular adhesion problem<sup>41</sup>, the evolutions of the traction force with respect to the contact area, determined by the ratio of  $R_A$  to  $R_b$ , are calculated and depicted under different  $R_b/R_p$  values (Extended Data Fig. 9b). The traction force first increases and then decreases as the contact area reduces, with a maximum value of  $F_t^{\max}$  dependent on the value of  $R_b/R_p$ . We further evaluated the relationship between  $F_t^{\max}$  and  $R_b/R_p$  as  $F_t^{\max} \approx \pi R_p^2 P_{\text{atm}} \phi_s$ , where the non-dimensional function  $\phi_s = \left(\frac{R_b}{R_p}\right)^2 - 1$  measures the ratio of the annular wetted area of the base to the sectional area of the pillar (Extended Data Fig. 9c).

### Energy analysis

To formulate the energy condition for the occurrence of freezing droplet ejection, we conducted a theoretical analysis to estimate the energy supply and energy consumption, respectively. The energy supply comes from the elastic energy stored in the spring-like pillar. At the impendent moment before the freezing droplet detaches from the SES, the stored elastic energy is calculated as  $E_e = \frac{1}{2} F_t \delta_{\text{cr}} = \frac{P_{\text{atm}}^2 V_p}{2K_c} \phi_s^2$ , where  $\delta_{\text{cr}} = \frac{P_{\text{atm}} h_p}{K_c} \phi_s$  is the critical compression of the pillar at which the repelling force  $F_t$  is equal to the maximal traction force  $F_t^{\max}$ . The inherent damping dissipation of the pillar accounts for  $(1 - \xi)E_e$ . Here  $\xi$  denotes the rebound resilience of the pillar, which was measured to be around 50% for the PDMS we applied (Extended Data Table 1). After detaching from the SES, the freezing droplet remains in contact with the pillar. When the droplet separates from the pillar, a negative pressure ( $-P_{\text{atm}}$ ) will be generated inside, resulting in a suction force  $\pi R_p^2 P_{\text{atm}}$  in between. The separation work is estimated as  $E_s = \pi R_p^2 P_{\text{atm}} (h_p - \delta_{\text{cr}}) = V_p P_{\text{atm}} [1 - \frac{P_{\text{atm}}}{K_c} \phi_s]$ . As a result, the kinetic energy gained by the freezing droplet for ejection is calculated as  $E_k = \xi E_e - E_s$ .

### Minimum water droplet volume for preventing water depletion

To avoid running out of liquid water during the freezing process, we consider the most conservative scenario, in which the volume expansion of the residual water at the beginning of the last cycle is enough to expel the pillar out from the ice shell. That is,

$$0.09V_d (1 - \phi_0^*) \geq V_p.$$

Substituting  $\phi_0^* = 1 - 1.96\mu^{-\frac{3}{5}} + \mu^{-1}$  into the above expression, we have

$$0.09V_d (1.96\mu^{-3/5} - \mu^{-1}) \geq V_p.$$

Recalling  $\mu = 0.09 \frac{V_d}{V_p} \frac{K_c}{\sigma_i}$ , the above inequality can be rewritten as

$$\frac{V_d}{V_p} \geq 2.07 \left( \frac{K_c}{\sigma_i} \right)^{-1} \left[ \left( \frac{K_c}{\sigma_i} + 1 \right) \right]^{5/2}.$$

This gives rise to a minimum requirement for the initial volume of the water droplet to prevent water depletion during the whole process before the separation of the ice shell and pillar. By some basic mathematical manipulations, the relationship between  $\frac{V_d}{V_p}$  and  $\frac{K_c}{\sigma_i}$  given by the above inequality can also be rewritten in an inverse form as

$$\frac{K_c}{\sigma_i} \leq 0.408 \left( \frac{V_d}{V_p} - 11.11 \right)^{0.732} + 0.667.$$

By multiplying both sides of the above inequality by  $\sigma_i/P_{\text{atm}}$ , which is presumably taken as  $\sigma_i/P_{\text{atm}} = 3$ , we can get

$$\frac{K_c}{P_{\text{atm}}} \leq 1.224 \left( \frac{V_d}{V_p} - 11.11 \right)^{0.732} + 2.$$

## Data availability

All data are available in the Article and its Supplementary Information. Source data are provided with this paper.

## Code availability

Code is available from the corresponding authors upon reasonable request.

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## Author contributions

Z.W. and H.Z. conceived the research. Z.W. and H.Y. supervised the research. H.Z., Y.J. and C.W. prepared the samples. H.Z., Y.J., C.W., and Z.X. designed and carried out the experiments. H.Z., S.Y., S.G. and F.L. analyzed the data. H.Y., H.Z. and W.Z. developed the theoretical modeling. Z.W., H.Y., W.Z., H.Z. and S.W. wrote the manuscript. All authors reviewed and approved the data.

## Competing interests

The authors declare no competing interests.

## Additional information

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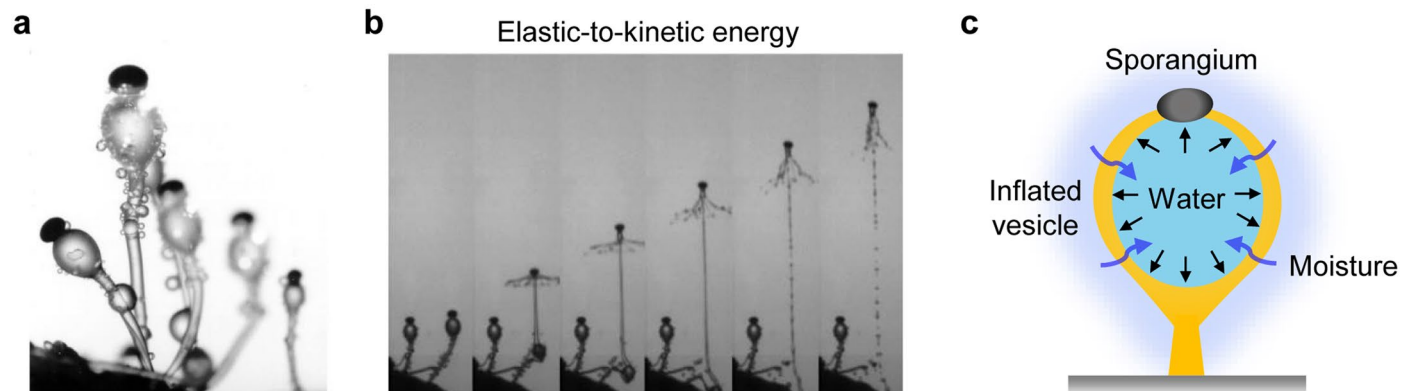
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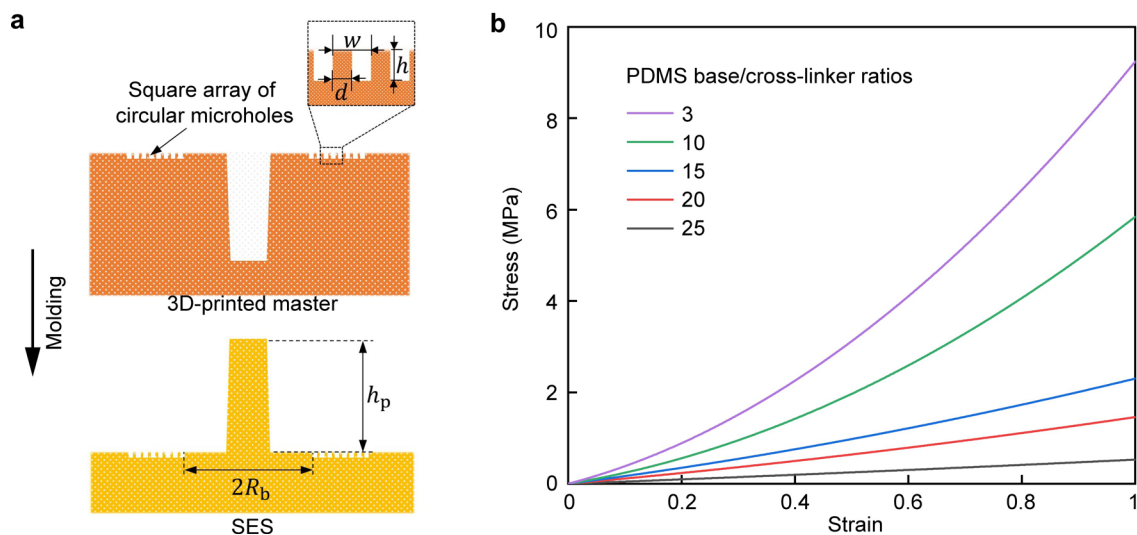
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**Extended Data Fig. 1 | Energy conversion for spore dispersal in the fungus (*Pilobolus kleinii*).** **a**, Photograph of the fungus (*Pilobolus kleinii*). Reproduced with permission from<sup>36</sup>. Copyright 2016 Elsevier. **b**, Snapshots showing the

spore dispersal behavior of the fungus driven by the conversion of elastic-to-kinetic energy. Reproduced with permission from<sup>36</sup>. Copyright 2016 Elsevier. **c**, Schematics of the mechanism of the energy conversion process in the fungus.

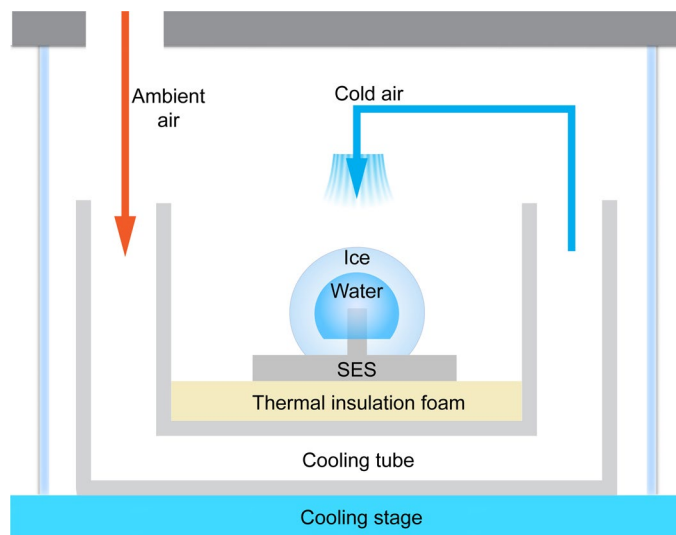


**Extended Data Fig. 2 | Fabrication and mechanical characterization of the SES.** **a**, Schematic illustration of the fabrication process of the SES. We fabricate the SES using the cast-molding method based on the 3D-printed masters.  $d$  and  $h$ , the diameter and height of the micropattern;  $w$ , the center-to-center spacing

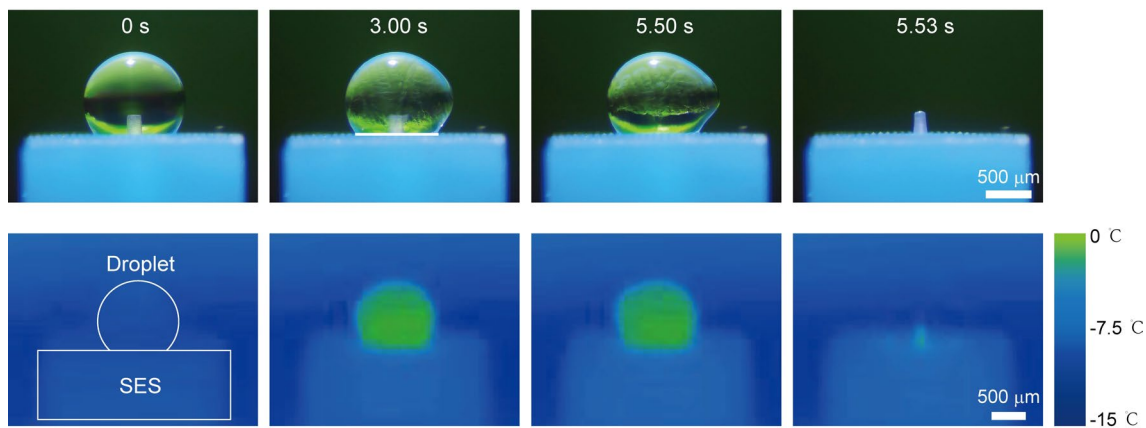
between the micro patterns;  $h_p$ , the height of the pillar;  $R_b$ , the radius of the smooth base. **b**, Compression tests of SES made of PDMS with different base/cross-linker ratios by weight. The compressive modulus ( $K_c$ ) equals the stress value when the strain reaches 1.



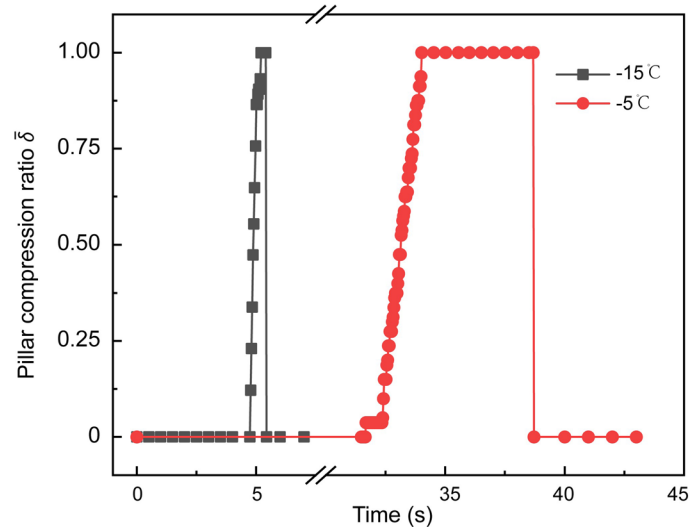
**Extended Data Fig. 3 | Contact angle characterization.** Contact angle of the water droplet on the SES.



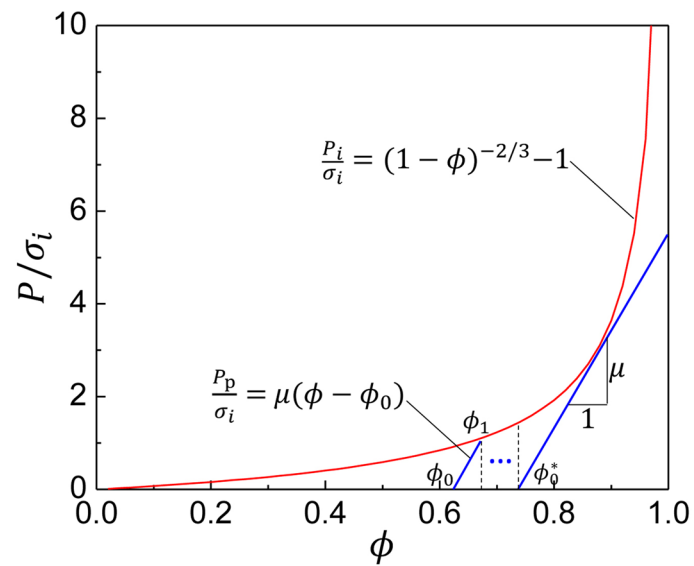
**Extended Data Fig. 4 | Schematic of the experimental setup for the freezing droplet ejection.** The droplet on the SES is cooled down by the chilly air circulated at -1 m/s within an isothermal environmental chamber ( $-15^{\circ}\text{C}$ ).



**Extended Data Fig. 5 | Optical and thermographic imaging of the water droplet freezing process on the SES sample.** The uniform temperature distribution within the SES illustrates that the droplet is cooled down by the circulated chilly air in the isothermal chamber.

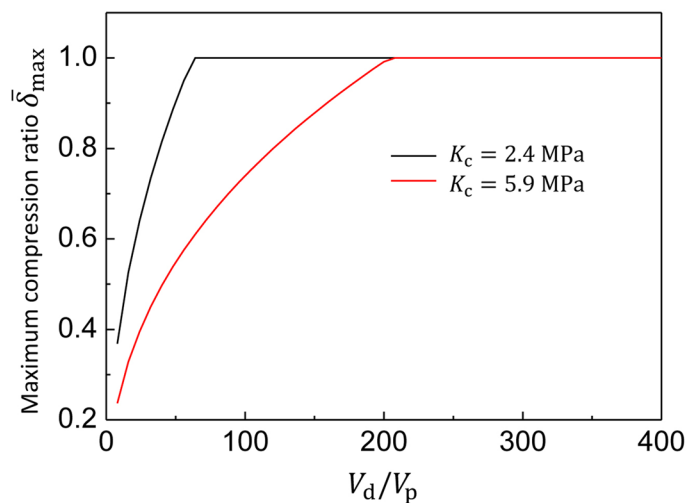


**Extended Data Fig. 6 | Influence of freezing conditions on pillar dynamics.** Real-time compression ratio of the pillar under two different freezing conditions,  $-15^{\circ}\text{C}$  and  $-5^{\circ}\text{C}$ .

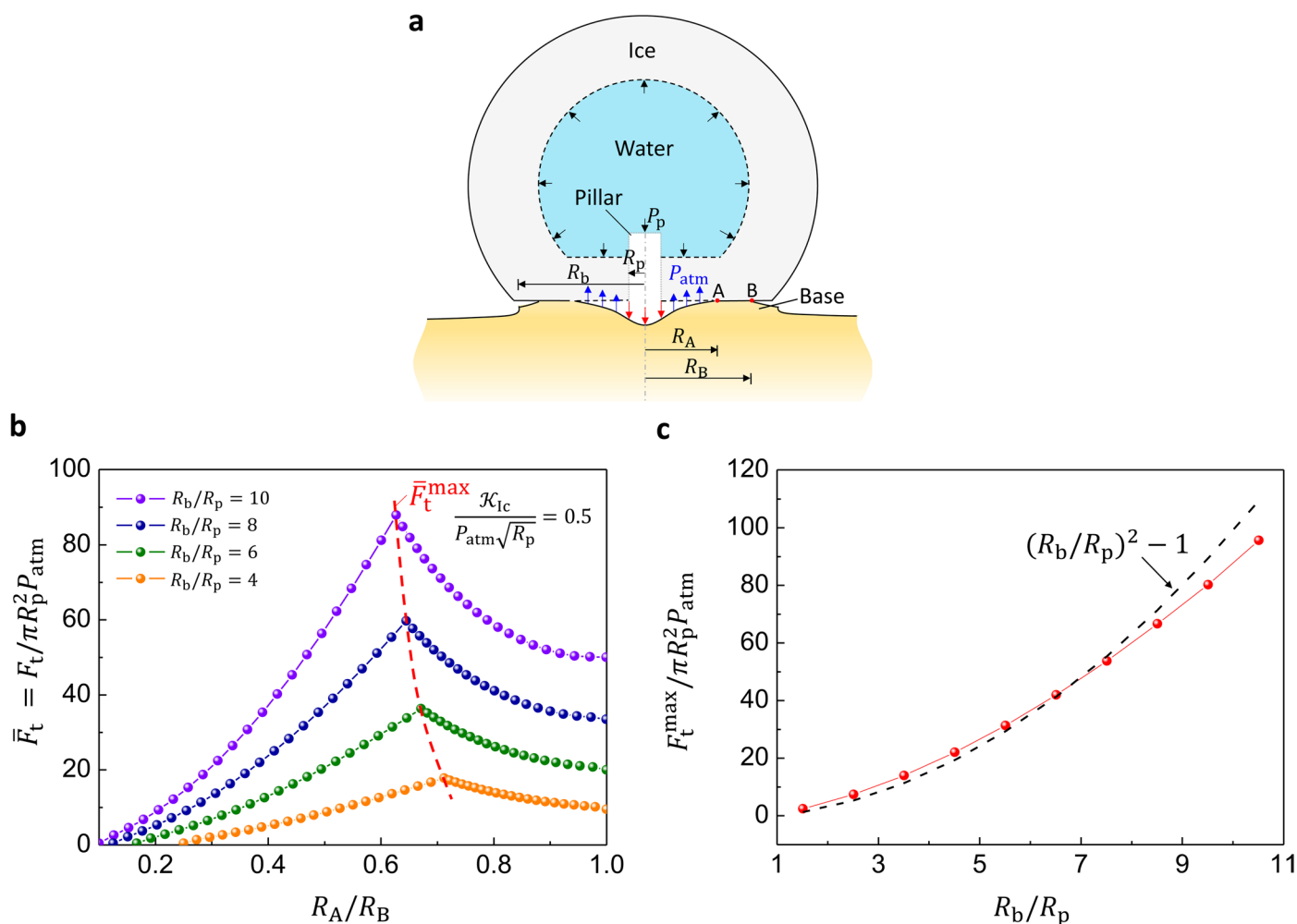


**Extended Data Fig. 7 | The dynamic competition between the pressure-bearing capacity of the ice shell  $P_i$  and the internal pressure  $P_p$ .** Here  $\phi_0^*$  denotes the critical value of the fraction of the solidified water ( $\phi_0$ ), whereby the ice shell will not fracture in the subsequent compression-rebound cycle.





**Extended Data Fig. 8** | The maximal compression ratio  $\bar{\delta}_{\max}$  as a function of the volume ratio of a droplet to a pillar  $V_d/V_p$  under different  $K_c$ . It can be found that  $\bar{\delta}_{\max}$  first increases as  $V_d/V_p$  increases and then reaches a plateau of 1. Here, the fracture strength of ice is taken as  $\sigma_i = 0.3$  MPa. Additionally,  $\bar{\delta}_{\max}$  exhibits a negative dependence on  $K_c$ . For a given  $V_d/V_p$ , the higher  $K_c$  the lower  $\bar{\delta}_{\max}$ .



**Extended Data Fig. 9 | Theoretical modeling for determining the maximum traction force between the freezing droplet and the base.** **a**, Schematics showing a snapshot of the separation process between a freezing droplet and the base of SES. **b**, The variation of the normalized traction force ( $F_t/\pi R_p^2 P_{\text{atm}}$ ) with the radius ratio between the inner and outer contact edges for different  $R_b/R_p$ . Here, the normalized fracture toughness of the interface was assumed as

$\frac{\mathcal{K}_{\text{Ic}}}{P_{\text{atm}}\sqrt{R_p}} = 0.5$  with  $\mathcal{K}_{\text{Ic}}$  being the critical stress intensity factor for interfacial delamination. **c**, Dependence of the maximum normalized traction force (the peak values on the curves in **b**) on the radius ratio between the base and pillar ( $R_b/R_p$ ).

**Extended Data Table 1 | The mechanical characterization of the SES made of PDMS with different base/cross-linker ratios by weight**

PDMS base/cross-linker ratio (by weight)	Compressive modulus $K_c$ (MPa)	Rebound resilience $\xi$
3	9.2	55%
10	5.9	51%
15	2.4	44%
20	1.7	45%
25	0.6	44%